

# High Efficiency Hydraulic Pump-Motors Employing Partial Stroke Piston Pressurization



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# **Hydraulic power - applications**



Image taken from: https://www.bobcat.com/loaders/compact-track-loaders/models/t590/photos-videos#lightbox-t590-m2-grapple-hay



Image taken from: https://www.caseih.com/northamerica/en-us/products-skid-steers/wheeled\_skid\_steer

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# **Existing Technology**



Image taken from: http://www.artemisip.com/our-products/industrial-hydraulic-pump

### Artemis Digital Displacement

- Two valves per piston
- Electronic sensors and controls
- Expensive

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### **Our approach: Hydro-Mechanical Control**

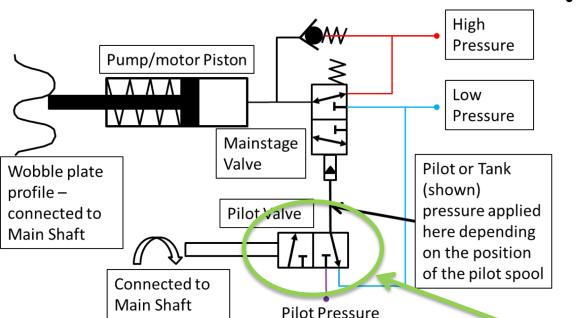
- Most discrete-piston implementations use electromagnetic valves:
  - 2 valves per piston
  - Artemis; others

#### Advantages of mechanical control – valve timing connected to shaft

- **Robust**: No solenoids/wires/power electronics to fail on each piston
- **Cost:** No controllers for each piston
- **Simple**: only 1 control input needed
- Mechanical and hydraulic power: No need for electrical supply
- Fast and repeatable timing: speed scales up with pump speed

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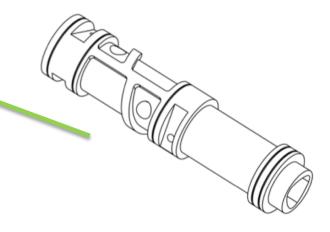
Concept



- Two stage
  - 2D Rotary valve pilot stage
  - Spool valve main stage

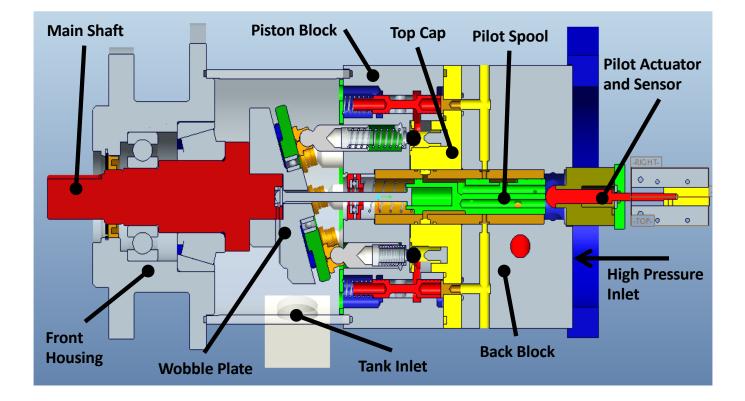
Pilot stage = 2 Degree-of-freedom valve:

- Rotation with shaft
- Translation: adjust displacement

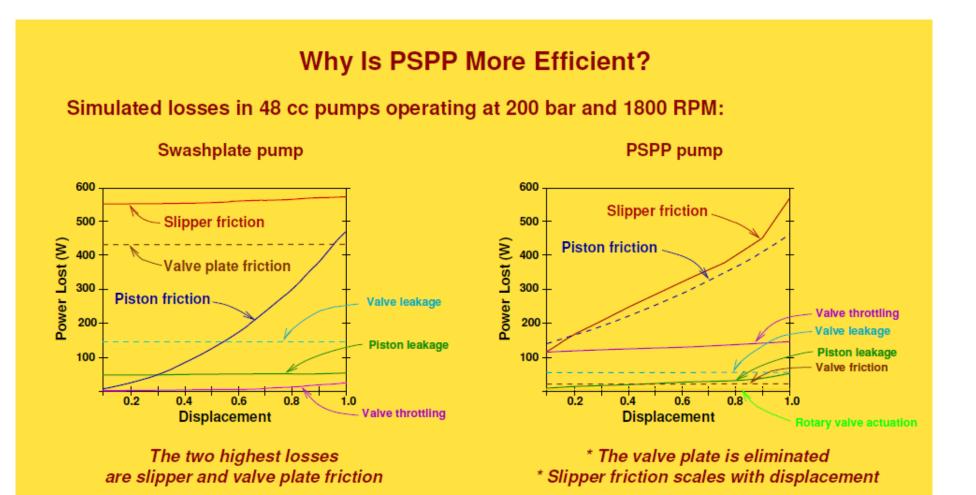


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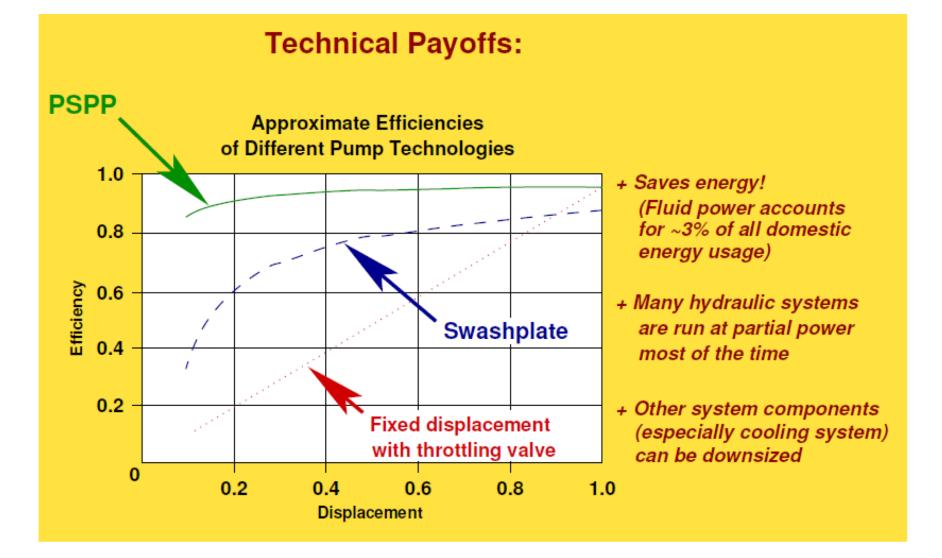
# **Cutaway CAD model**



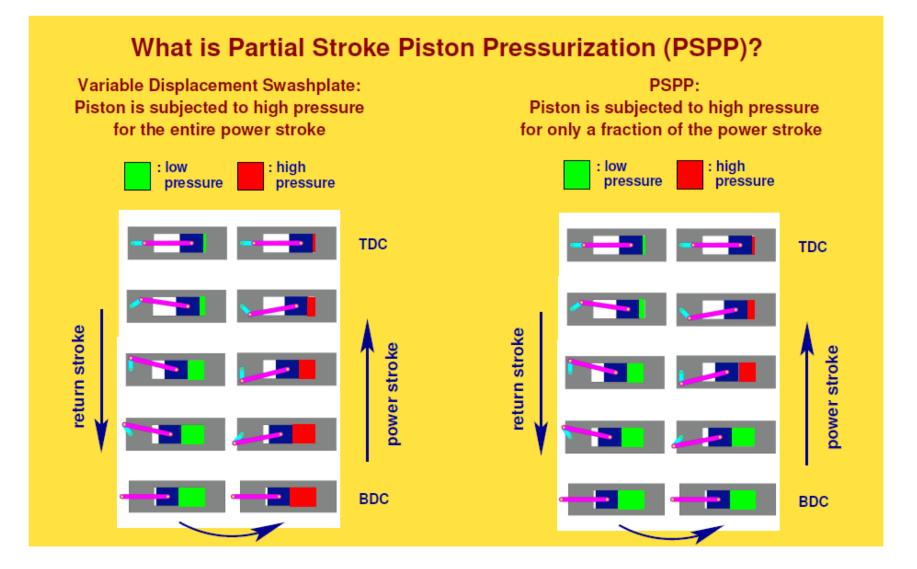
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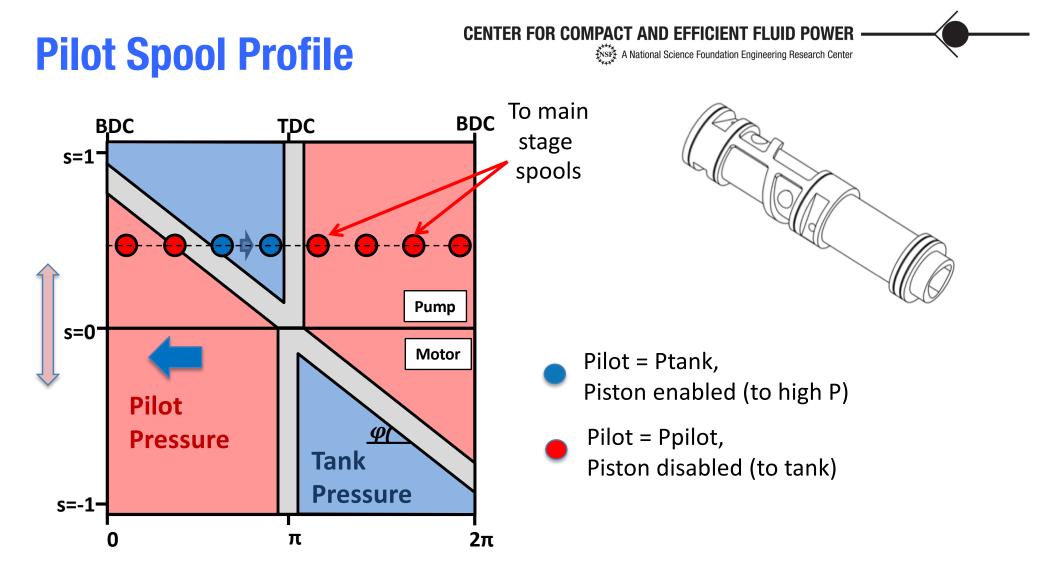


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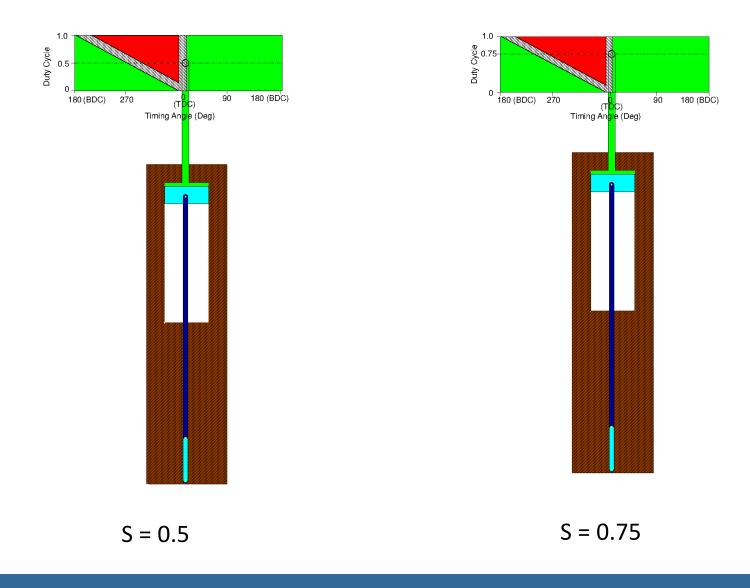
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- Spool translation: changes stroke
  - Pump starts stroke late and motor ends stroke early
    - Ideal for reducing compressibility loss

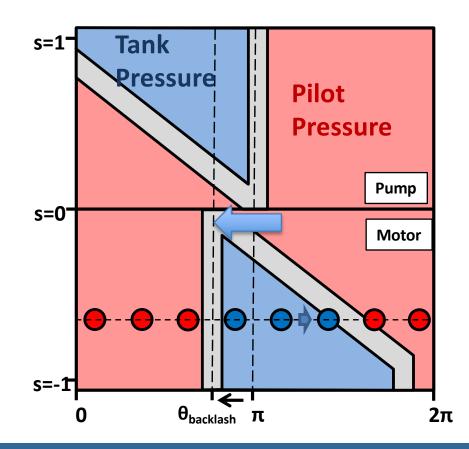
## **Animation**



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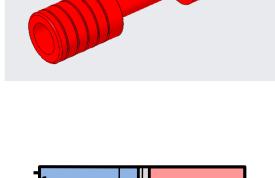
### **Pre-compression backlash**

- Shift motor timing so the transition between low and high pressure in the piston chambers precedes TDC position
  - add backlash between shaft and wobble plate



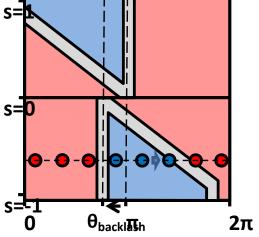
## Improvements

- Radial clearance on mainstage valves has been reduced from ~40 microns to ~4 microns
- Backlash angle setting now accessible from outside of pump case
- New mechanism for manually setting duty cycle of rotary spool



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• Rebuilt test stand and data acquisition system

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• Able to see pumping when run at 1000psi

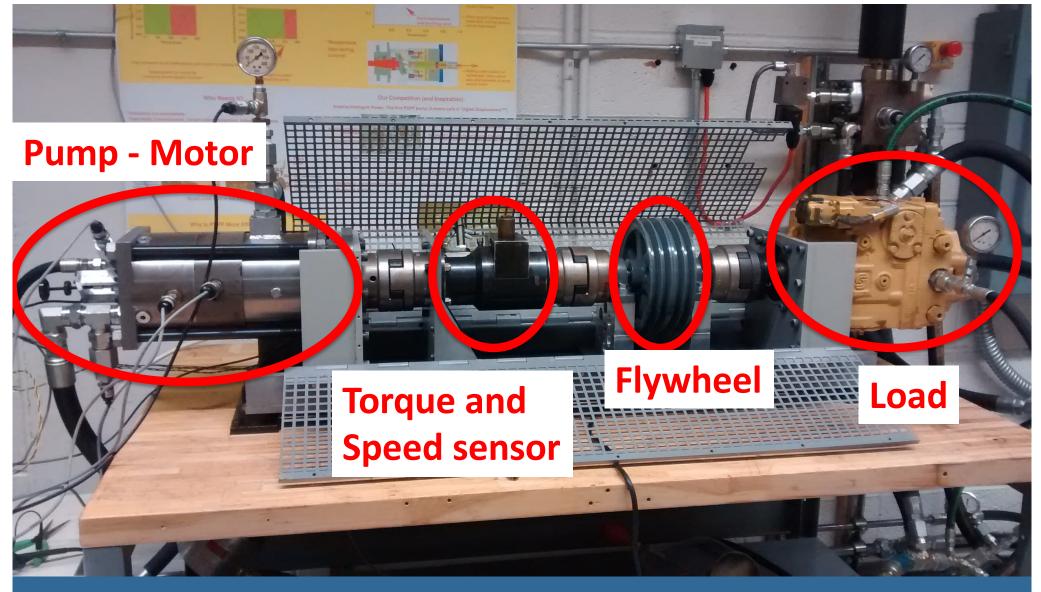
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## **Test Stand**



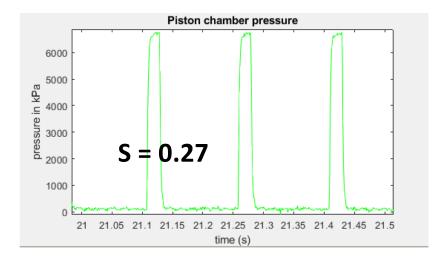
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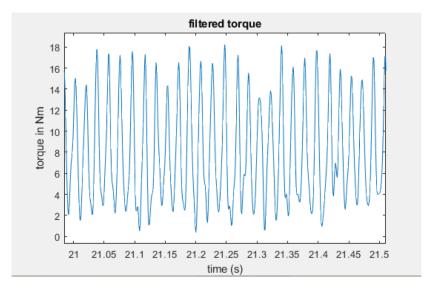
## **Test Stand**

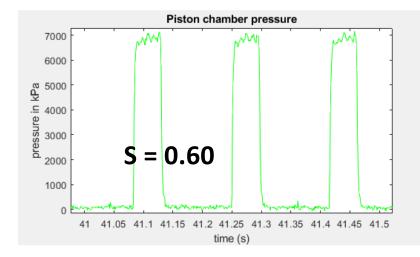


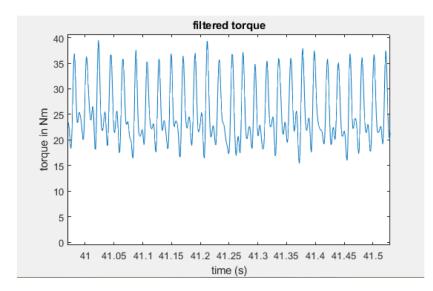
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# **Results – pump efficiency data**









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# **Results – pump efficiency data**

- S = 0.27, measured efficiency = 6%
- S = 0.60, measured efficiency = 10%

Potential reasons for low efficiency

- Pilot line leakage (mainstage valves could be moving too slowly causing throttling losses)
- Backlash timing set incorrectly
- Could be run at higher pressure

# Conclusions

 Discrete piston control using mechanical valving has advantages of simplicity and ruggedness

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- Our design uses:
  - Rotary pilot, 3-way spool main stage, partial stroke
- Measured efficiency is low, but should be improved by removing pilot leakage and setting timing angle